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Research on Simulation of Neuro-fuzzy Control Strategy of Eddy Current Retarder^①

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Abstract

A simulation model of eddy current retarder is designed, including 1/4 vehicle model, eddy current retarder model, tire model. Based on the PID and neuro-fuzzy control technique, two kinds of control algorithm on constant speed control heavy truck are designed. On the built simulation system, the control algorithm is simulated and tested. The results show that the neuro-fuzzy controller can implement high precision and strong robust control effects on control of eddy current retarder system.

Index Terms: Eddy Current Retarder; Heavy Truck Model; Neuro-fuzzy Control

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1. Introduction

Eddy Current Retarder (ECR) is a device that transfers kinetic energy generated by vehicle into thermal energy which will emit afterwards. Based on electromagnetic theory, ECR can realize deceleration and retardation. It can keep the main brake remaining in the cooling state in order to work effectively in emergencies. The use of retarder can significantly reduce the wear of the main brake, extend life expectancy to 4-10 times the original, which will greatly reduce operating costs, improve the usability of vehicle.

Most of the currently-existing retarder adopts hierarchical control relays[1], so the braking torque generated is hierarchical and not continuous. Thus the braking torque generated differs from the actual demands, hard to meet the braking requirements[2].

In this paper the neuro-fuzzy control technique is utilized to develop a new stepless retarder control method. A retarder and vehicle model is built to simulate the process. The result shows that the neuro-fuzzy control method can achieve better eddy current retarder stepless control and make vehicles go downhill smoothly at a constant speed, and have strong robustness and adaptability.

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2. Simulation Model of ECR

2.1. Vehicle Model

According to the requirement of simulation, a 1/4 vehicle kinematic model in ramp way is built, and rolling resistance is neglected during braking. Assuming the vehicle goes straight, without considering the problem of tire lateral force, the longitudinal dynamics equation is:

$$F_j = F_i - F_f - F_w \quad (1)$$

Where F_j is vehicle deceleration resistance, F_i is sliding force, F_f is adhesion, F_w is air resistance.

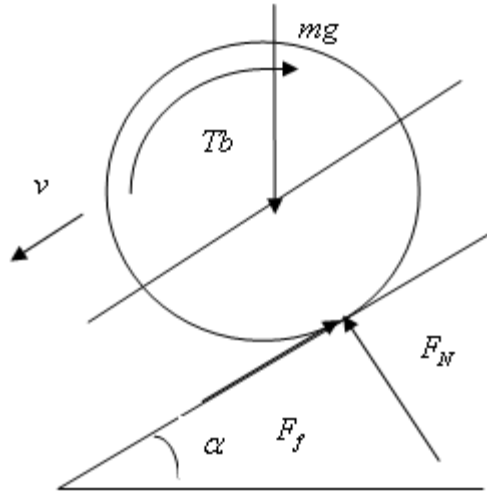


Fig. 1. Single wheel truck model in rampway

Vehicle deceleration resistance, sliding power, traction, air resistance can be calculated by the following formula:

$$F_j = -M\dot{v} \quad (2)$$

$$F_f = \mu Mg \quad (3)$$

$$F_w = \frac{C_d A v_a^2}{21.15} \quad (4)$$

$$F_i = Mg \sin \alpha \quad (5)$$

Where μ is the adhesion coefficient between the wheels and the ground, which is a function of tire slip ratio, $\mu = f(s)$. C_d is air resistance coefficient. v is vehicle speed. ω is angular velocity for the wheels. α is ramp angle. M is the vehicle quality.

Tire model under stress of heavy truck in braking:

$$J\dot{\omega} = T_b - F_f r \quad (6)$$

Where J is the rotational inertia of the wheels, $\dot{\omega}$ is the angular deceleration of the wheels, F_f is rolling resistance, T_b is the braking torque effected on the wheels, r is the radius of the wheels.

$$T_b = T_\omega \cdot i_0 \quad (7)$$

T_ω is the retarder braking torque, i_0 is the drive ratio of the final reducer in transmission.

2.2. The dynamic model of ECR

Braking torque is the main standard measured of eddy current retarder. However, it's a very difficult task to accurately calculate the braking torque. Its institutional structure is not very complicated, but the braking torque is affected by many factors. Currently, the calculate of eddy current retarder braking torque adopt a method of combining simplifying and testing calculation. In this paper, on the basis of access to reference[3], the formula of calculating of eddy current retarder braking torque can be expressed as follows:

$$T_1 = 90\sqrt{2}N_p (\rho\mu_0)^{\frac{2}{3}} (NI)^2 \pi d^4 \sqrt{\omega} \quad (8)$$

$$T_2 = \sqrt{\mu_r} \arcsin\left(\frac{d}{2R_l}\right) (16\pi\rho l_\varepsilon + \sqrt{2}k_e \mu_0 \sqrt{\frac{2\rho}{\omega\mu_0\mu_r} \frac{\pi d^2}{4} \omega})^2 \quad (9)$$

$$T = \frac{T_1}{T_2} \quad (10)$$

Where k_e is conversion coefficient, which usually $k_e = 1.5$. ρ is resistivity of disk, $\rho = 1.5 \times 10^{-7} \Omega \cdot m$. σ is conductivity of disk. μ_0 is the vacuum permeability, $\mu_0 = 4\pi \times 10^{-7} H/m$. N is the excitation coil turns, taken 320. μ_r is relative permeability of disk, low carbon steel is usually taken 200. ω is angular frequency for the magnetic field. N_p is the pole pairs. l_ε is air gap, taken 1.4mm.

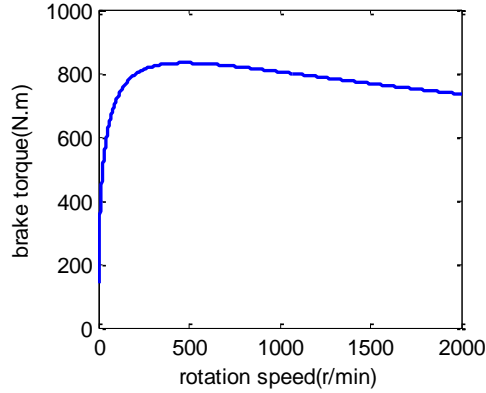


Fig. 2. Brake torque of ECR model

$$d \text{ is core diameter, } d = 101.5\text{mm} . \quad \omega = \frac{2\pi N_p n}{60} .$$

2.3. Tire Model

Tire model adopts the magic formula to calculate the tire lateral force and aligning torque, the magic formula is shown as follows:

$$Y(x) = D \sin(C \arctan(Bx - E(Bx - \arctan(Bx)))) \quad (11)$$

Where Y can be the longitudinal force , which can also be a lateral force or aligning torque, etc. the variables X can express the longitudinal slip rate or slip angle of the tire, respectively, under different circumstances. B, C, D, E are determined successively by the tire vertical load and camber angle, which are the constant related with the surface. By changing these parameters , different road adhesion coefficient can be simulated. The parameters in this magic formula used in this paper are shown as follows:

$$D = 0.9, C = 2.2, B = 6.0, E = 0.98.$$

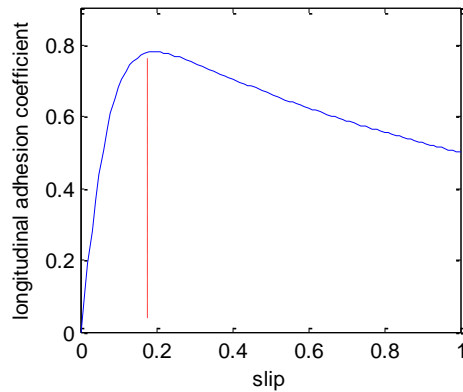


Fig. 3. Longitudinal adhesion coefficient

3. Retarder Control Strategy

Adaptive Neuro-Fuzzy Inference System in the MATLAB is a system which combines the neural network theory with T-S fuzzy reasoning[4]. It can be built the fuzzy Inference system (FIS) with an adaptive modeling method, based on plenty of data, because of the establishment of the FIS using neural network is the result of data processing.

At first, the PID control is the method to control data in this article, and then take the results as the training data, according to the algorithm provided by MATLAB to calculate and built a two-dimensional T-S model. The vehicle speed deviation and speed deviation change rate is selected as the input and the excitation current as the output to realize the control of the constant speed when vehicles go downhill. The neural fuzzy control structure shown in Fig.4.

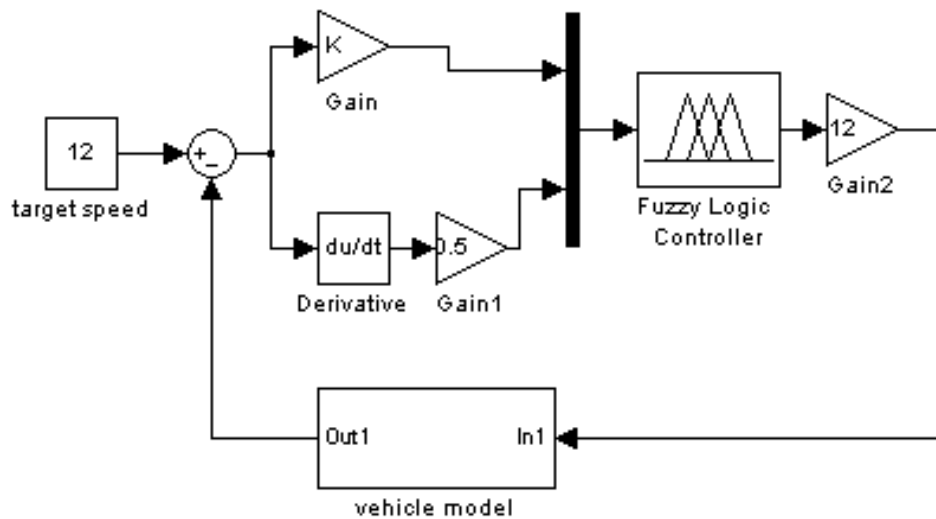


Fig. 4. Neuro-fuzzy controller

Through neural network training, there are three fuzzy rules:

if e is e1 and ec is ec1 then

$$u1 = -11.65u1 - 5.821u2 + 15.62$$

if e is e2 and ec is ec2 then

$$u2 = -11.41u1 - 5.908u2 + 15.94$$

if e is e3 and ec is ec3 then

$$u3 = -11.42u1 - 5.879u2 + 15.44$$

4. Ramp Constant Speed Control Simulation Model

According to the requirement of simulation[5], a vehicle ramp constant speed control model which shown in the Fig.5 is built, mainly including PID and neuro-fuzzy control model, eddy current retarder model, tire model, 1 / 4 vehicle model.

Eddy current retarder is expressed as(8-10), tire model adopt the magic formula (11), PID and fuzzy neural controller is the two-step controlled or also be individually controlled.

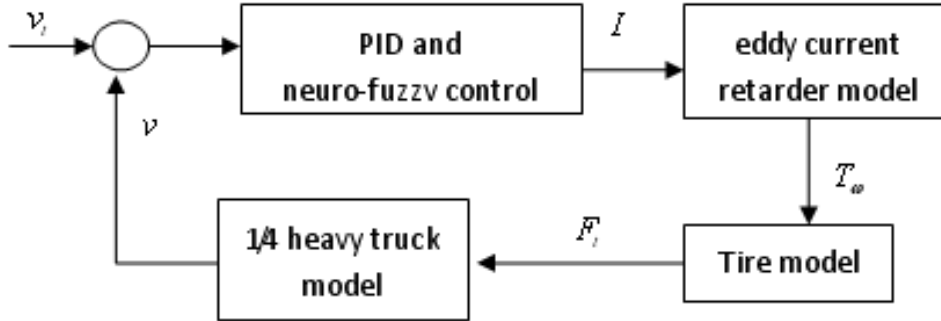


Fig. 5. Simulation model of heavy truck with ECR

5. Simulation Results and Analysis

In the MATLAB / simulink simulation circumstances, the constant control of the vehicle simulation in rampway model is built. Set up vehicles and the eddy current retarder simulation parameters are as follows: $m = 7000\text{kg}$, $r = 0.52\text{m}$, $J = 20\text{kg} \cdot \text{m}^2$, $A = 6\text{m}^2$, $i_0 = 5.833$, $\eta = 0.85$, $C_d = 0.3$, the results shown as follows:

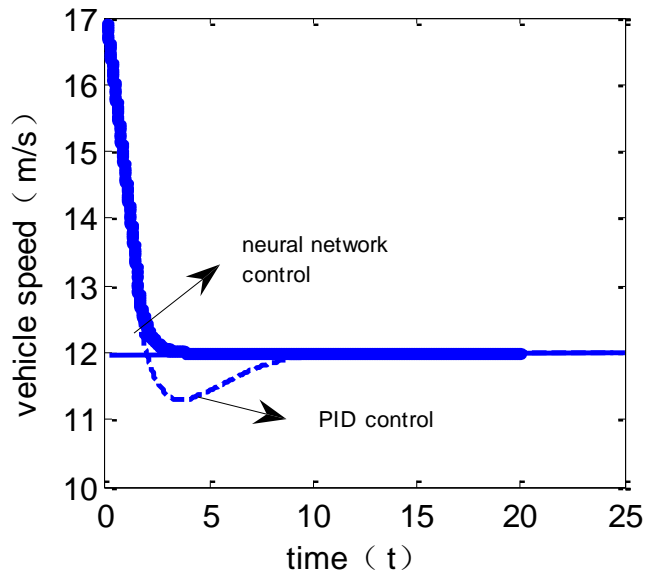


Fig. 6. The comparison between neural network control and PID control

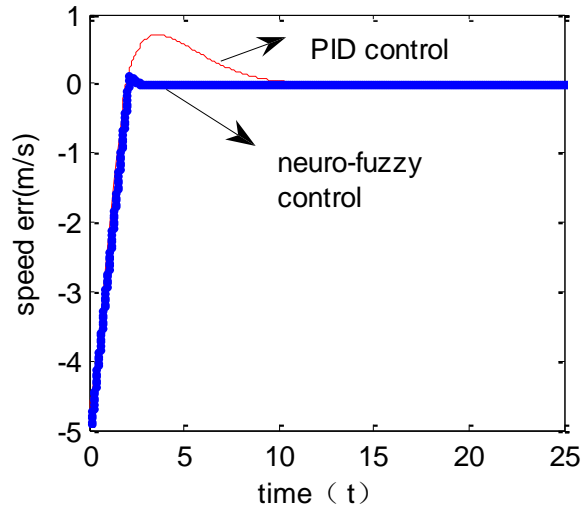


Fig. 7. The change of deviation in the two control method

A conclusion can be drawn that it is appropriate to achieve constant speed control of the eddy current retarder in braking when go downhill. This smart constant speed control can improve vehicle's safety and driver's comfort.

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